



## DESIGN AND ANALYSIS OF PERMANENT MAGNET DIRECT DRIVE FOR HOIST OF 1 T ELECTRIC OVERHEAD TRAVELLING CRANE

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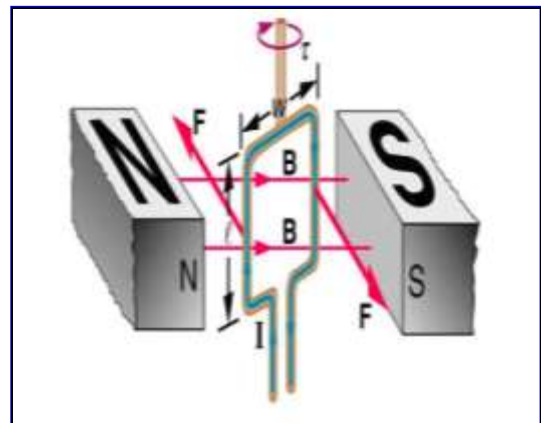
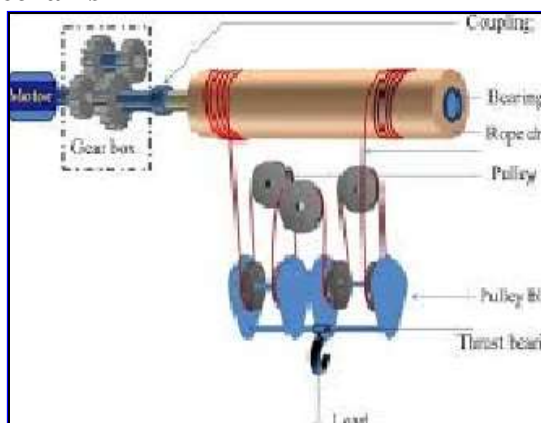
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**ABSTRACT** – The conventional hoisting mechanism used in industries consists of complicated four way transmission system of motor- gearbox -coupling- drum and rope assembly. Which is subjected to frequent failures due to failure of any of these above mentioned components .Here we have proposed a novel concept of implementing the Permanent Magnet Direct Drive (PMDD) in hoisting mechanism for 1 T Electric overhead Travelling(EOT) Crane. Implementation of PMDD can completely eliminate use of Gearbox; which we need for high torque and low RPM. PMDD is designed and analyzed for successful implementation in hoisting mechanism. The Designed and Analyzed PMDD will be able to give the high torque at low RPM for 1 T EOT crane. The PMDD offers many other significant advantages over conventional mechanism like almost zero maintenance, no lubrication, physical Isolation, less vibration, more reliability less power consumption.

### INTRODUCTION

The conventional mechanism used in EOT crane is consisting of Motor- Gearbox- coupling- drum and rope assembly. This research has been carried to because of the frequent failures in conventional hoisting mechanism like failure of gearbox, failure of coupling, frequent lubrication and maintenance, excessive vibration and excessive noise. Due to vibration the parts may deteriorate slowly and ultimately leads to failure It will lead to a great monetary loss too especially in railway yards, steel industries etc. In order to accommodate the frequent failures; the shop floor must need excess inventory which may not lead to stop the working of EOT crane for various crucial operations.

**Figure 1. Conventional Hoisting Mechanism** **Figure 2. Proposed Principle for Hoisting Mechanism**



The novel idea of the Permanent magnet motor drive to implement in hoisting mechanism is inspired by the concept of using the natural, free and sustainable energy of Permanent Magnets

(PM). With innovation in the field of PM and availability of magnetic materials like Samarium-cobalt (Sm-Co) and Neodymium-iron-boron (Nd-Fe-B) are the PM which offers very good Properties for PM and gives excellent high temperature performance.

After doing the thorough literature survey PMDD was identified as the best suitable option to use in hoist. After that Stator and Rotor which are the most important part of the PMDD is designed and analyzed to use in hoist of 1 T EOT crane. We may use the designed and analyzed PMDD to replace the gearbox without affecting the performance. This will lead to more significant advantages in a long run. We are able to get the higher torque at lower RPM with designed stator and rotor assembly of PMDD. Due to that we can totally eliminate the conventional gearbox used in hoisting mechanism to get the higher torque and required less RPM [1].

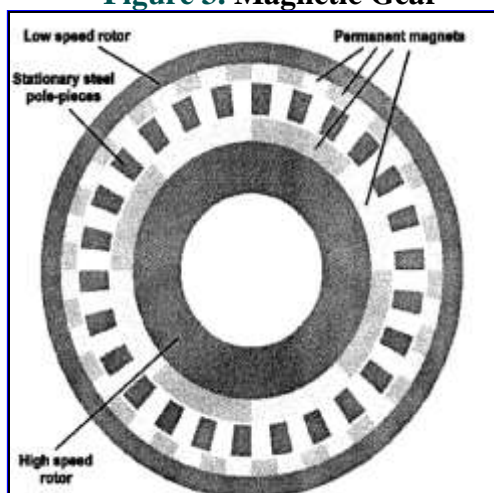
The remainder of this paper is organized as follows: Section 2 briefly presents relevant literature survey. Section 3 explains the design of hoist for 1 T hoist. Section 4 provides the analysis of PMDD. Section 5 concludes the paper.

## LITERATURE SURVEY

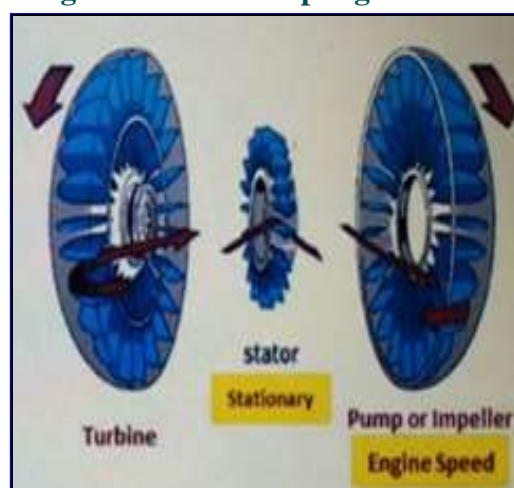
A thorough literature survey has been done to figure out the most suitable alternative for conventionally used hoisting mechanism and to replace the gearbox. Few alternatives studied are Magnetic Gears, Fluid Coupling and PMDD with Variable Frequency Drive.

Magnetic gear (MG) offers significant advantages of reduced acoustic noise, minimum vibration, free from maintenance, improved reliability, inherent overload protection, and physical isolation. The concept of magnetic gearing into the permanent-magnet machines, leads to achieve low-speed high-torque direct-drive operation. Low utilization and poor performance of ferrite permanent magnet (PM) material made it impossible to be widely used in industry. Until the high-performance neodymium iron boron (Nd-Fe-B) these converted MGs simply replaced the slots and teeth of iron core by N-poles and S-poles of PMs, respectively. Unlike the converted MGs, the Coaxial MG has a higher torque density, because all the PMs simultaneously contribute to torque transmission. In view of the coaxial structure, the CMG can be artfully integrated with a high-speed outer rotor PM brushless machine to constitute a composite electrical machine named as the magnetic-gear permanent-magnet (MGPM) machine, which can achieve low-speed high-torque driving while providing high torque density. The MPG with six magnetic planet gears exhibits nearly  $100\text{kNm/m}^3$  [1, 2, and 3].

**Figure 3. Magnetic Gear**



**Figure 4. Fluid Coupling**

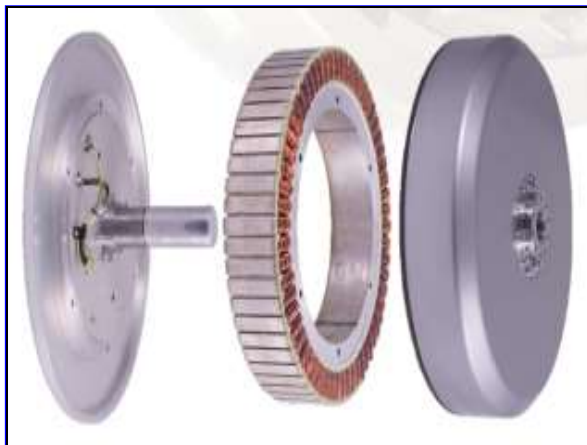


A hydrodynamic or "hydrokinetic" device called a fluid coupling or hydraulic coupling is used to transfer rotating mechanical power. It is also widely used in maritime and industrial machine drives, where regulated start-up at different speeds is essential to prevent shock loading of the power transmission system. [4].

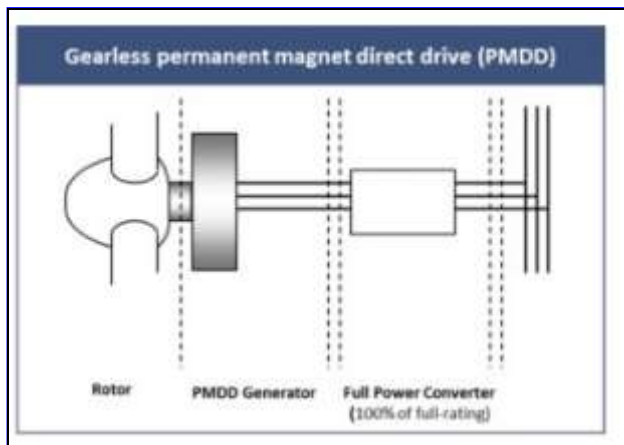
PMDD with Variable Frequency Drive is studied for the proposed concept and which is proved to be the best suitable for hoisting mechanism to get high torque at low RPM. The key concept is to eliminate the four part transmission structure of the shaft, reducer, drum, and motor to realize the direct drive in order to address the problems in the crane industry [5-7].

There is no intermediate mechanical transmission in motor drive technology. Both the linear motion component with a linear motor serving as the core drive element and the rotary motion component with a torque motor serving as the core drive element are applications of direct drive technology.

**Figure 5. PMDD**



**Figure 6. Line Diagram PMDD**



Currently, elevators, machine tools, belt machines, mines, wind power generation, and other industries employ direct drive technology extensively. It is more usual to obtain direct motor drive by doing away with the intermediary transmission as low speed and high torque motor technology continues to advance.

### DESIGN OF PMDD USED IN HOIST FOR 1 T EOT CRANE

Permanent Magnet Direct Drive Motor is attempted to design here. The basic variables used in the Design Method are [8-10];

**Table 1.** Variables used in the Design

Design Variables		Corresponding basic variables	
d	Air gap diameter	d	Air gap diameter
l	Stator length	l	Stator length



$h_s$	Slot height	$h_s$	Slot height
$\tau_P$	Pole pitch	$\tau_P$	Pole pitch
$J_B$	Current density	$J_B$	Current density
$B_{\delta 0}$	Peak air gap flux density	$h_m$	Magnet height
$B_{d0}$	Peak teeth flux density	$b_d$	Tooth width

Constant and fixed relations		Corresponding basic variables	
$B_{ys}$	Peak Stator yoke flux density	$h_{ys}$	Stator yoke height
$B_{yr}$	Peak rotor yoke flux density	$h_{yr}$	Rotor yoke height
$h_{s1} = 1 \text{ mm}$	Tooth tip height	$h_{s1}$	Tooth tip height
$h_{s2} = 4 \text{ mm}$	Slot wedge height	$h_{s2}$	Slot wedge height
$b_{s1} = 3 \text{ mm}$	Slot Opening	$b_{s1}$	Slot opening
$h_i = 1 \text{ mm}$	Insulation Thickness	$h_i$	Insulation Thickness
$\delta = 0.001 d$	Mechanical Air gap	$\delta$	Mechanical Air gap
$q = 1$	No. of slots per pole	$\tau$	Slot Pitch



	& phase		
$b_m=0.7\tau_p$	Magnet Width	$b_m$	Magnet Width

**Design Equations [8-10]**

To determine number of poles,

$$N = \frac{120 \times f_s}{P} \quad (1)$$

where,  $N = \text{RPM}$

$f_s = \text{frequency}$

$P = \text{Number of poles}$

The diameter and pole pitch, determine the pole pairs,

$$P = \frac{\pi d}{2 \cdot \tau_p} \quad (2)$$

where,  $p = \text{Number of pole pairs}$

$\tau_p = \text{Pole pitch, } m$

$d = \text{Air gap diameter, } m$

The total number of slots on stator is

$$Q = 2 p m q \quad (3)$$

where,  $p = \text{number of pole pairs}$

$q = \text{number of slots per pole}$

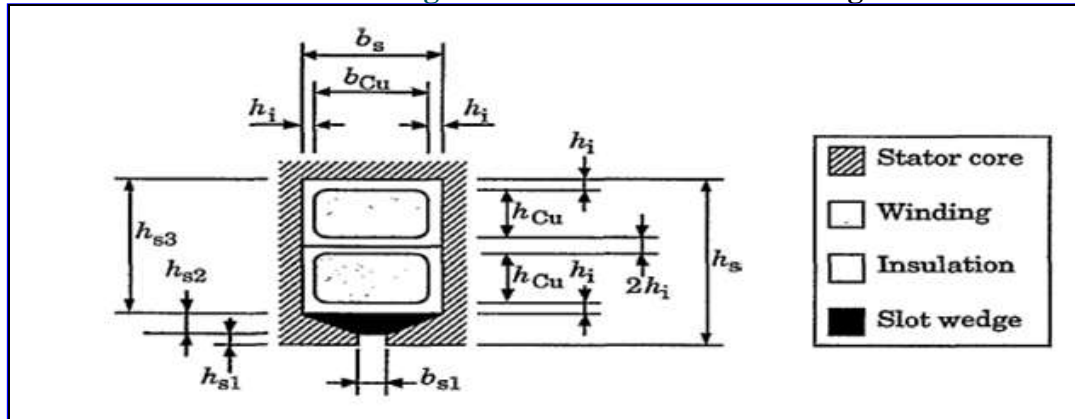
$m = \text{number of phase}$

The Slot Pitch is;

$$\tau = \frac{\tau_p}{m q} \quad (4)$$

The slots and the two layer windings are shown in Figure: 7. The slot is described by its depth  $h_s$  and its width  $b_s$ .

**Figure 7. Basic Variables in Design**



The slot width can be calculated from the slot pitch  $\tau$  and tool width  $b_d$  as:

$$b_s = \tau - b_d \quad (5)$$

where,  $b_s$  = slot width

$b_d$  = tooth width

$\tau$  = slot pitch

$h_s$  = slot depth

The slot opening  $b_{s1}$  is assumed to be 3mm, the tooth tip height  $h_{s1}$  = 1mm, and the slot wedge height  $h_{s2}$  = 4mm. The winding height is  $h_{s3}$

$$h_{s3} = h_s - h_{s1} - h_{s2} \quad (6)$$

The conductor height  $h_{cu}$  and width  $b_{cu}$  are determined by the winding height, slot width and the coil insulation thickness  $h_i$ ;

$$h_{cu} = \frac{h_{s3} - 4h_i}{2} \quad (7)$$

$h_{cu}$  = conductor height

$h_i$  = insulation height

$$b_{cu} = b_s - 2h_i \quad (8)$$

$b_{cu}$  = conductor width

For a three phase machine for which the magnet cost per torque should be kept low, the ratio of magnet width to pole pitch should be between 0.6 and 0.9 (Lampola et. Al 1996a). In the proposed design the magnet width is kept at 0.7 times the pole pitch, i.e.,

$$b_m = 0.7 \tau_p \quad (9)$$

$b_m$  = magnet width



As the winding is a full pitch winding, the winding pitch  $W$  is;

$$W = \tau_p \quad (10)$$

The end winding length is assumed to be

$$l_b = 2 W \quad (11)$$

The Equivalent core length is approximated by

$$l_e = l + 2 \delta \quad (12)$$

The useful iron length is

$$l_u = k_{Fes} l \quad (13)$$

where,  $k_{Fes}$  = iron fill factor

The frequency at rated speed is

$$f = p n_N \quad (14)$$

where,  $p$  = pole pair

The iron gap should be small to minimize the amount of Permanent Magnet needed .The mechanical stiffness and the thermal expansion limits the minimum air gap which can be used is,

$$\delta = 0.001 d \quad (15)$$

The slot opening is narrow compared with the air gap, so carter factor will be 1.The outer diameter of the stator  $d_{se}$

$$d_{se} = d + 2 h_s + 2 h_{ys} \quad (16)$$

Approximate total length of the stator

$$l_{total} = l + 3 W \quad (17)$$

### Calculated Dimensions

By using the above dimensions for we do get following values;

**Table 2.** Calculate Dimensions

Design parameter	Calculated value	Design parameter	Calculated value
N	15 – 20 rpm	lb	20 mm

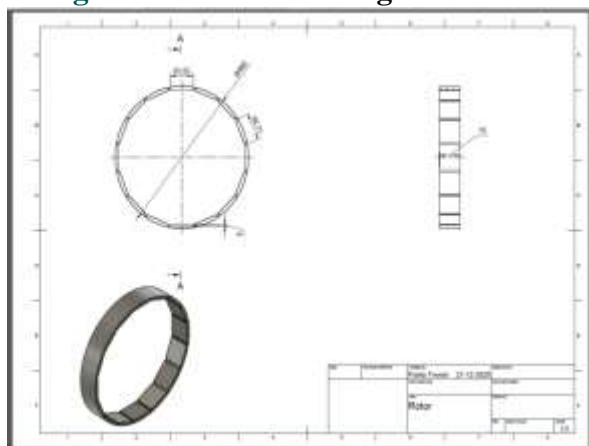


P	32 nos.	le	75 mm
Pp	16 nos.	dse	485mm
$\tau_p$	10 mm	hs3	20mm
$\tau$	3.33 mm	hcu	8 mm
bs1	1mm	bcu	0.165 mm
hs1	4mm	bm	7 mm
w= $\tau_p$	10 mm		

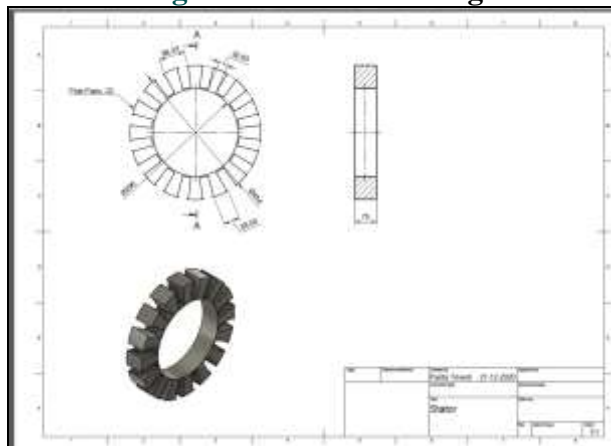
**Modeling of Stator Rotor as per the Calculate Dimensions**

Modeling is done in fusion i360 for stator, rotor and Assembly of stator and Rotor.

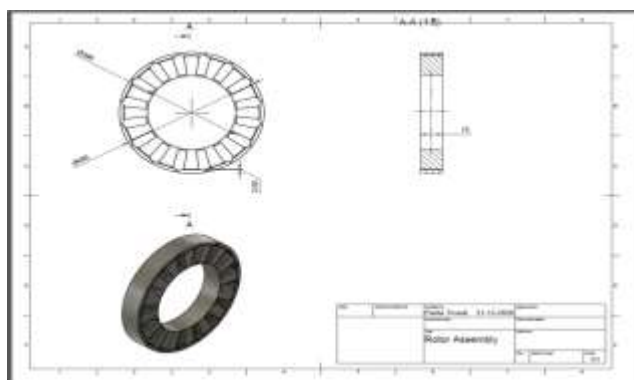
**Figure 8. Detail Drawing of a Stator**



**Figure9. Detail Drawing of Rotor**



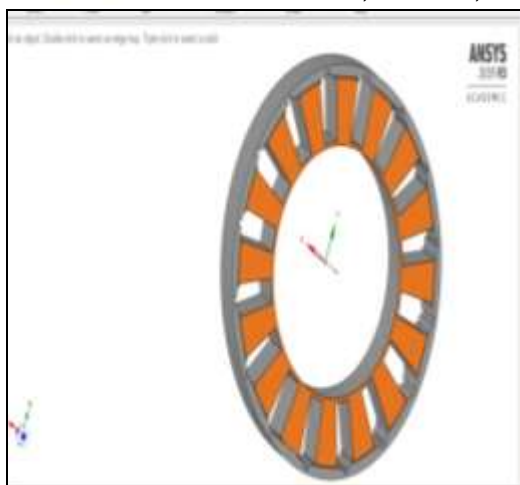
**Figure10. Assembly of Stator and Rotor**



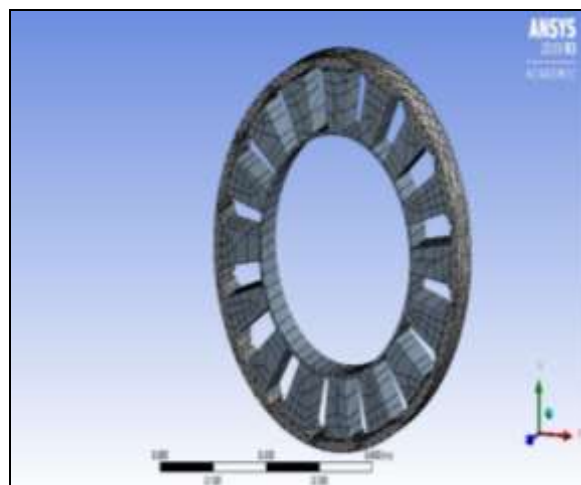
**ANALYSIS OF PMDD**

Analysis is done in Ansys workbench 2019 R3 **Meshing : Meshing with Nodes: 17350 Elements: 4585**





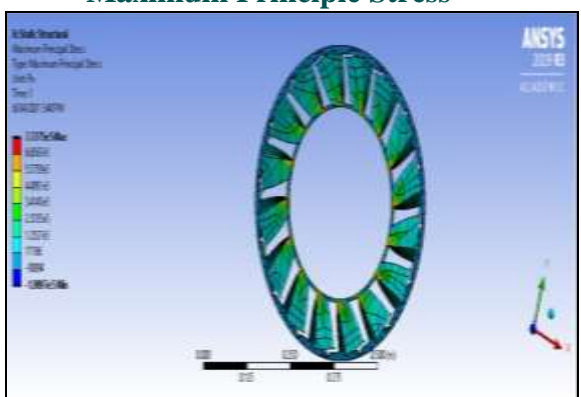
**Equivalent stress**



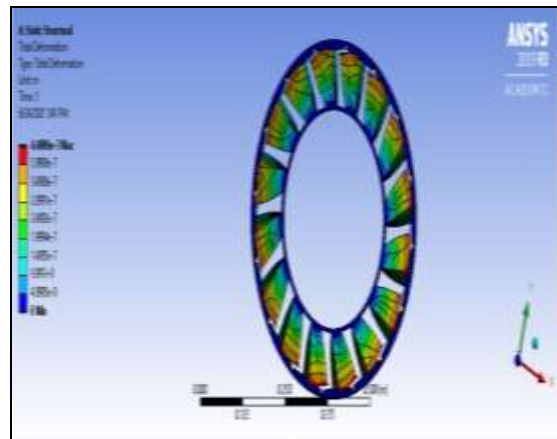
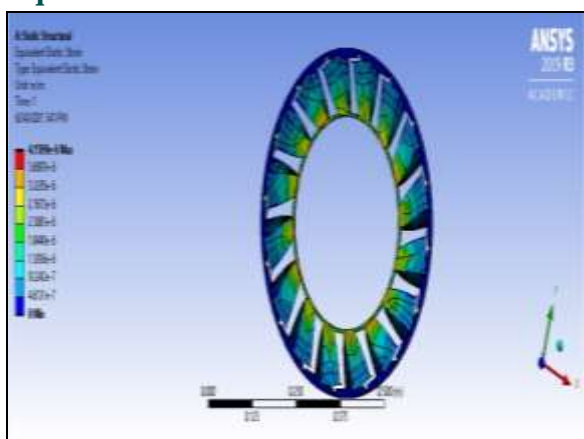
**Maximum Principle Stress**



**Equivalent Elastic strain**



**Deformation**



**Summary of Analysis**

**Table 2.** Summary of Analysis in Ansys

Type of Analysis	Structural	Unit
Equivalent Von-Mises Stress	$8.27 \times 10^5$	Pa
Maximum Principal stress	$7.73 \times 10^5$	Pa
Equivalent Elastic Strain	$4.15 \times 10^{-6}$	m/m




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Total Deformation	$4.49 \times 10^{-7}$	m
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**Comparison diagram of failure rate of the Lifting Mechanism**

Failure rate of Conventional mechanism and the proposed mechanism is given as below [11]

**Table 3. Failure rate comparison**

S N	Part	Traditional Hoist		Proposed Hoist	
		Failure Rate %	Main Failure	Failure Rate %	Main Failure
1	Gearbox	22	Broken teeth, poor oil quality, insufficient oil quantity, oil leaks, gear heating, and vibration	0	0
2	Motor	13	Motor heating, a short circuit, a cracked installation base, a failed fan, etc.	5	Motor heating, a short circuit, a cracked installation base, a failed fan, etc.
3	Shaft	6	Fractures, joint deformity, and loose joints	0	0
4	Coupling	20	A loose lubrication nut, an unusual sound, wear, and deformation	0	0
5	Brake	18	Deformation loosely mounted brake wheel. unusual braking torque mechanisms and a decline in braking torque	7	Bolt loosening, oil pipe leakage, spring ageing, etc.
6	Rope	5	rope fracture, skip rope groove, and rope brake	5	rope fracture, skip rope groove, and rope brake
7	Lifting control system	16	Older electric components, unsecured terminals, and burned inverter	16	Older electric components, unsecured terminals, and burned inverter
Total		100		33	

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According to the above table, the suggested lifting mechanism has a failure rate that is 67 percent lower than the traditional lifting mechanism, which accounts for 45 percent of the failure rate of the entire crane.

### Conclusion

This paper reports the first attempt to implement the newer concept of Permanent Magnet Direct Drive motor to use in hoist used for 1 Tonnage Electric overhead travelling crane. PMDD motor is designed and analyzed for 1 T EOT crane. For further research we may go for higher capacities of EOT crane, Tower crane and other material handling equipments.

### ACKNOWLEDGEMENT

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